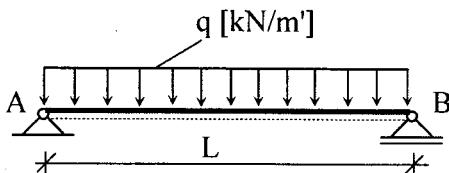


## Vežba br. 12a

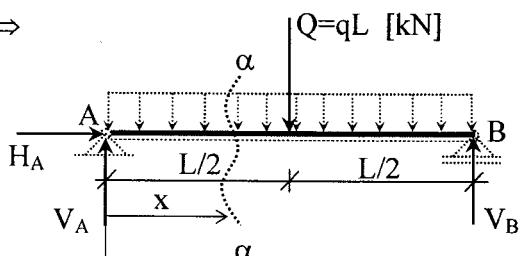
### Sile u presecima. Prosta greda opterećena kontinualnim opterećenjem

Odrediti sile u presecima za datu prostu gredu i nacrtati dijagrame momenata savijanja, transverzalnih sila i normalnih sila.

1.



$$(A5) \Rightarrow$$



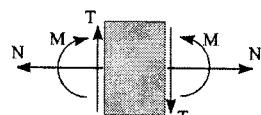
Uslovne jednačine ravnoteže:

$$\sum X = 0 \Rightarrow H_A = 0, \quad (1)$$

$$\sum Y = 0 \Rightarrow V_A + V_B - Q = 0, \quad (2)$$

$$\sum M_A = 0 \Rightarrow V_B \cdot L - Q \cdot \frac{L}{2} = 0, \quad (3)$$

$$(1) \Rightarrow H_A = 0, \quad (3) \Rightarrow V_B = \frac{Q}{2} = \frac{qL}{2}, \quad (2) \Rightarrow V_A = \frac{Q}{2} = \frac{qL}{2}.$$



Analitički izrazi za moment savijanja, transverzalnu i normalnu силу на прстож греди AB (пресек  $\alpha - \alpha$ ), где се  $x$  креће у границама од нула до  $L$ , глase:

$$M(x) = V_A x - q x \frac{x}{2} = \frac{qL}{2} x - q \frac{x^2}{2},$$

$$T(x) = V_A - q x = \frac{qL}{2} - q x,$$

$$N(x) = 0,$$

a vrednosti momenata savijanja, transverzalnih i normalnih сила у крајњим тачкама A и B су:

$$\text{за } x=0 \rightarrow M_A = 0, \quad T_A = \frac{qL}{2}, \quad N_A = 0,$$

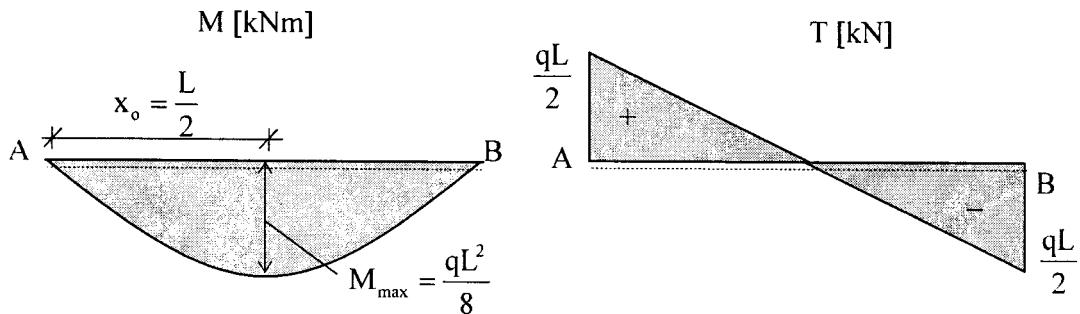
$$\text{за } x=L \rightarrow M_B = 0, \quad T_B = -\frac{qL}{2}, \quad N_B = 0.$$

Moment savijanja se menja po zakonu parabole drugog reda, čija strela može da se odredi iz uslova ekstremuma funkcije. Imajući u vidu diferencijalnu zavisnost između momenta savijanja i transverzalne силе у произвљеном пресеку гредног nosačа, maksimalni moment savijanja se nalazi у пресеку у коме је transverzalna сила једнака нули:

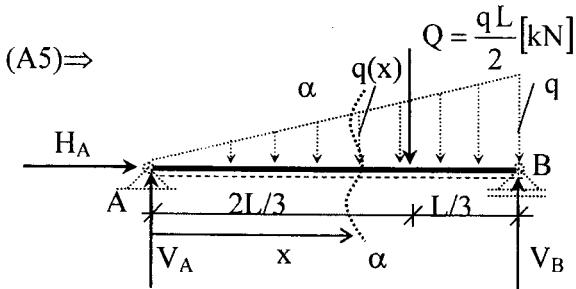
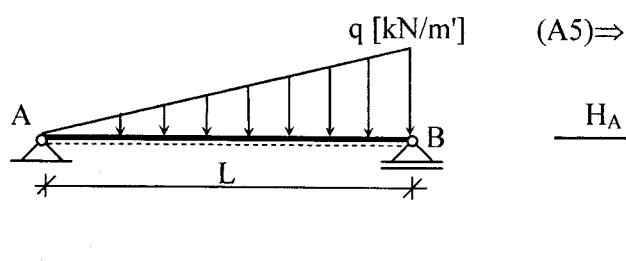
$$T(x) = \frac{dM(x)}{dx} = 0 \Rightarrow \frac{q}{2}(L - 2x_0) = 0 \Rightarrow x_0 = \frac{L}{2}.$$

Veličina tog maksimuma je:

$$M_{\max} = M_{(x_e = \frac{L}{2})} = \frac{qL}{2} \frac{L}{2} - q \frac{\left(\frac{L}{2}\right)^2}{2} = \frac{qL^2}{8}.$$



2.



$$\sum X = 0 \Rightarrow H_A = 0, \quad (1)$$

$$\sum Y = 0 \Rightarrow V_A + V_B - Q = 0, \quad (2)$$

$$\sum M_A = 0 \Rightarrow V_B \cdot L - Q \cdot \frac{2L}{3} = 0, \quad (3)$$

$$(1) \Rightarrow H_A = 0, \quad (3) \Rightarrow V_B = \frac{2Q}{3} = \frac{qL}{3}, \quad (2) \Rightarrow V_A = \frac{Q}{3} = \frac{qL}{6}.$$

Pri formiranju analitičkih izraza za momente savijanja i transverzalne sile potrebno je poznavati zakon promene kontinualnog opterećenja  $q = q(x)$  na rastojanju  $x$  od oslonca A. U ovom primeru to je prava linija, a njena jednačina se može odrediti pomoću sličnosti pravouglih trouglova iz kojih sledi proporcionalnost odgovarajućih stranica:

$$q : q(x) = L : x \Rightarrow q(x) = \frac{q x}{L}.$$

Analitički izraz za moment savijanja, gledajući levo od preseka i levi deo šablonu, glasi:

$$M(x) = V_A x - \frac{q(x)x}{2} \frac{x}{3} = \frac{qL}{6}x - \frac{\frac{q x}{L}x}{2} \frac{x}{3} = \frac{qL}{6}x - \frac{q}{6L}x^3.$$

a vrednosti momenata savijanja na početku i na kraju polja su:

$$\text{za } x = 0 \rightarrow M_A = 0,$$

$$\text{za } x = L \rightarrow M_B = 0.$$

Prvi izvod momenta savijanja po koordinati x je jednak transverzalnoj sili:

$$T(x) = \frac{dM(x)}{dx} = \frac{qL}{6} - \frac{q}{2L}x^2,$$

a vrednosti transverzalnih sila na početku i na kraju polja su:

$$x=0 \rightarrow T_A = \frac{qL}{6},$$

$$x=L \rightarrow T_B = -\frac{qL}{3}.$$

Kako transverzalna sila na jednom delu polja ima pozitivan, a na drugom negativan znak, to ukazuje na to da u polju AB postoji presek u kome je transverzalna sila jednak nuli. Ako je taj presek na rastojanju  $x_0$  od oslonca A, može se napisati:

$$\frac{qL}{6} - \frac{q}{2L}x_0^2 = 0,$$

odakle se određuje položaj traženog preseka :

$$x_0 = \frac{L\sqrt{3}}{3}.$$

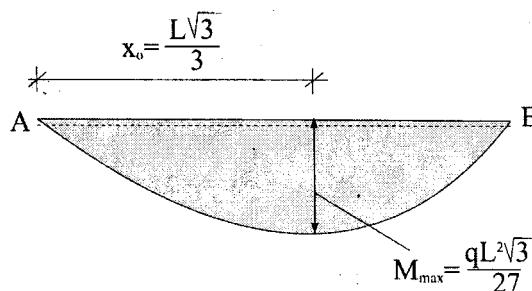
Pošto je:

$$T(x) = \frac{dM(x)}{dx},$$

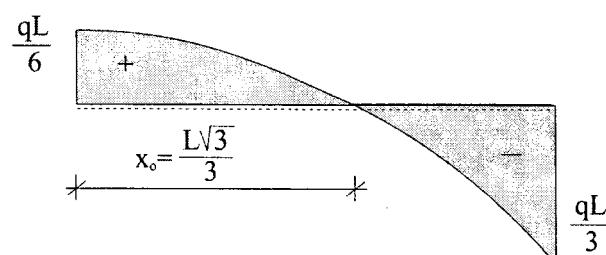
u tački  $x=x_0$  funkcija  $M(x)$  ima prevojnu tačku, to jest u tom preseku moment savijanja ima maksimalnu vrednost:

$$M_{\max} = M_{\left(x_0 = \frac{L\sqrt{3}}{3}\right)} = \frac{qL}{6} \frac{L\sqrt{3}}{3} - \frac{q}{6L} \left(\frac{L\sqrt{3}}{3}\right)^2 = \frac{qL^2\sqrt{3}}{27}.$$

Aksijalne sile su na celoj gredi jednake nuli.

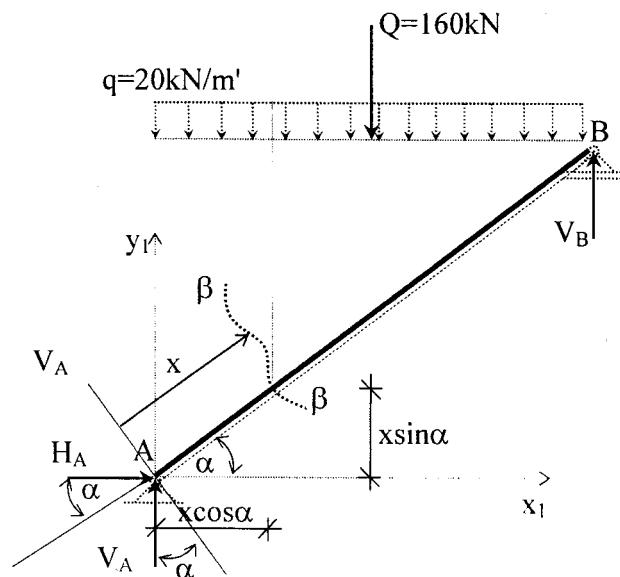
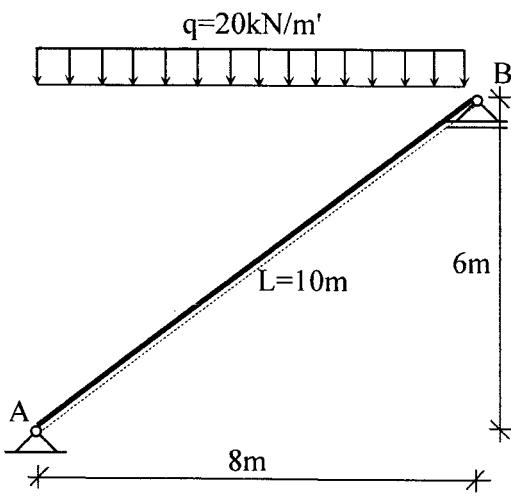


$M \text{ [kNm]}$



$T \text{ [kN]}$

3.



$$L = \sqrt{8^2 + 6^2} = 10\text{m}, \quad \sin \alpha = \frac{6}{10} = 0.60, \quad \cos \alpha = \frac{8}{10} = 0.80,$$

$$Q = qL \cos \alpha = 160\text{kN}.$$

Reakcije veza:

$$\sum X_1 = 0 \Rightarrow H_A = 0, \quad (1)$$

$$\sum Y_1 = 0 \Rightarrow V_A + V_B - Q = 0, \quad (2)$$

$$\sum M_A = 0 \Rightarrow V_B \cdot 8 - Q \cdot 4 = 0, \quad (3)$$

$$(1) \Rightarrow H_A = 0, \quad (3) \Rightarrow V_B = \frac{Q}{2} = \frac{qL \cos \alpha}{2} = 80\text{kN}, \quad (2) \Rightarrow V_A = \frac{Q}{2} = \frac{qL \cos \alpha}{2} = 80\text{kN}.$$

Analitički izrazi za moment savijanja, transverzalnu i aksijalnu silu u polju proste grede su:

$$M_\beta = V_A x \cos \alpha - Q(x) \frac{x \cos \alpha}{2} = \frac{qL \cos \alpha}{2} x \cos \alpha - q x \cos \alpha \frac{x \cos \alpha}{2} = \frac{qL \cos \alpha}{2} x \cos \alpha - \frac{q \cos^2 \alpha}{2} x^2.$$

$$T_\beta = V_A \cos \alpha - q x \cos \alpha \cos \alpha = \frac{qL \cos \alpha}{2} \cos \alpha - q x \cos^2 \alpha = \frac{qL}{2} \cos^2 \alpha - q x \cos^2 \alpha.$$

$$N_\beta = -V_A \sin \alpha + q x \cos \alpha \sin \alpha = -\frac{qL \cos \alpha}{2} \sin \alpha + q x \cos \alpha \sin \alpha.$$

Položaj i vrednost maksimalnog momenta u polju AB:

$$\frac{dM}{dx} = T = A \cos \alpha - q x_0 \cos^2 \alpha = 0 \Rightarrow x_0 = \frac{q L \cos^2 \alpha}{2 q \cos^2 \alpha} = \frac{L}{2} = 5\text{m}.$$

$$M_{\max} = \frac{q L \cos^2 \alpha}{2} \frac{L}{2} - \frac{q L^2}{2} \cos^2 \alpha = \frac{q L^2}{8} \cos^2 \alpha = \frac{20 \cdot 10^2}{8} \left( \frac{8}{10} \right)^2 = 160\text{kNm}.$$

Vrednosti momenata savijanja, transverzalnih sila i aksijalnih sila na početku i na kraju polja su:

$$M_A = 0, \quad M_B = 0,$$

$$T_A = V_A \cos \alpha = 80 \frac{4}{5} = 64 \text{kN},$$

$$T_B = \frac{qL}{2} \cos^2 \alpha - qL \cos^2 \alpha = -\frac{qL}{2} \cos^2 \alpha$$

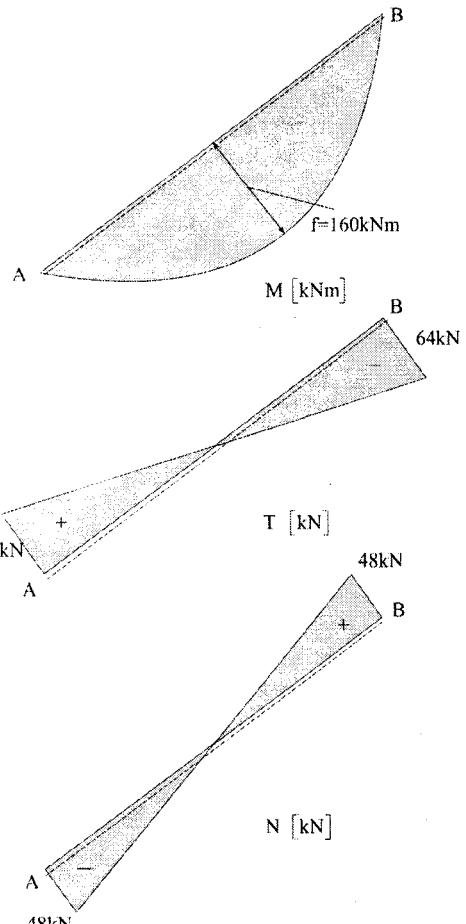
$$= -V_B \cos \alpha = -80 \frac{4}{5} = -64 \text{kN},$$

$$N_A = -\frac{qL \cos \alpha}{2} \sin \alpha = -V_A \sin \alpha$$

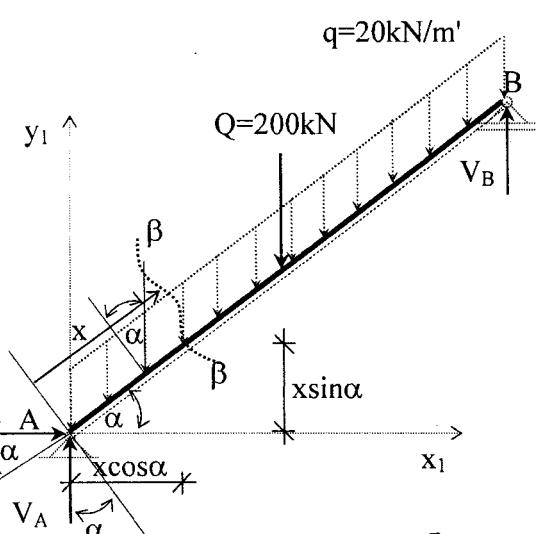
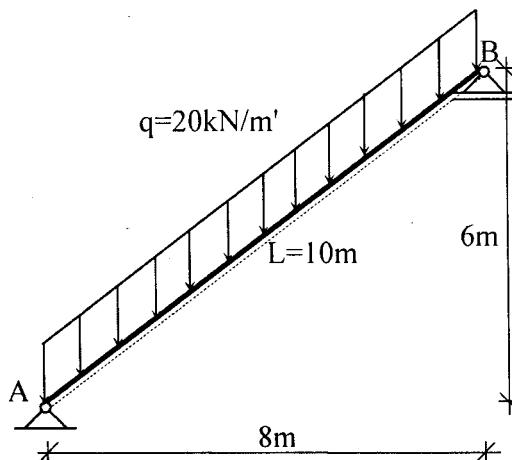
$$= -80 \frac{6}{10} = -48 \text{kN},$$

$$N_B = -\frac{qL \cos \alpha}{2} \sin \alpha + qL \cos \alpha \sin \alpha$$

$$= \frac{qL \cos \alpha}{2} \sin \alpha = V_B \sin \alpha = 80 \frac{6}{10} = 48 \text{kN}.$$



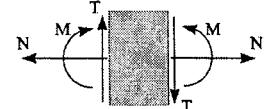
4.



Reakcije veza:

$$\sum X_1 = 0 \Rightarrow H_A = 0, \quad (1)$$

$$\sum Y_1 = 0 \Rightarrow V_A + V_B - Q = 0, \quad (2)$$



$$\sum M_A = 0 \Rightarrow V_B \cdot 8 - Q \cdot 4 = 0, \quad (3)$$

$$(1) \Rightarrow H_A = 0, \quad (3) \Rightarrow V_B = \frac{Q}{2} = \frac{qL}{2} = 100\text{kN}, \quad (2) \Rightarrow V_A = \frac{Q}{2} = \frac{qL}{2} = 100\text{kN}.$$

Analitički izrazi za moment savijanja, transverzalnu i normalnu silu u polju proste grede glase:

$$M_\beta = V_A x \cos \alpha - (q \cos \alpha) x \frac{x}{2} = \frac{qL}{2} x \cos \alpha - \frac{qx^2}{2} \cos \alpha.$$

$$T_\beta = V_A \cos \alpha - (q \cos \alpha) x = \frac{qL}{2} \cos \alpha - q x \cos \alpha,$$

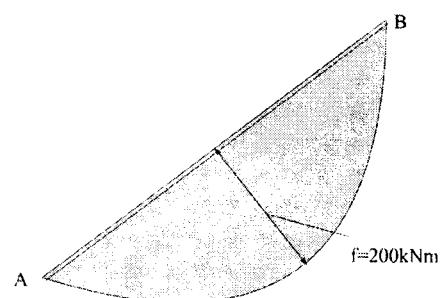
$$N_\beta = -V_A \sin \alpha + (q \sin \alpha) x = -\frac{qL}{2} \sin \alpha + q x \sin \alpha.$$

Položaj i vrednost maksimalnog momenta u polju AB:

$$\frac{dM}{dx} = T = V_A \cos \alpha - q x_0 \cos \alpha = 0$$

$$\Rightarrow x_0 = \frac{qL \cos \alpha}{2q \cos \alpha} = \frac{L}{2} = 5\text{m},$$

$$M_{\max} = \frac{qL}{2} \frac{L}{2} \cos \alpha - \frac{q \left(\frac{L}{2}\right)^2}{2} \cos \alpha = \frac{qL^2}{8} \cos \alpha \\ = \frac{20 \cdot 10^2}{8} \frac{8}{10} = 200\text{kNm}.$$

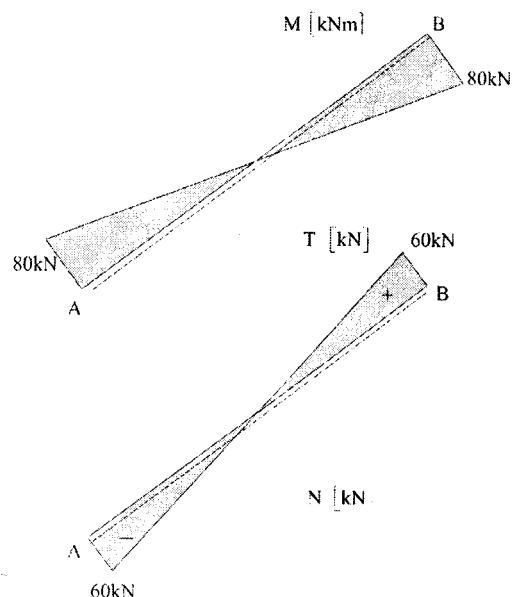


Vrednosti momenata savijanja, transverzalnih sila i aksijalnih sila na početku i na kraju polja su:

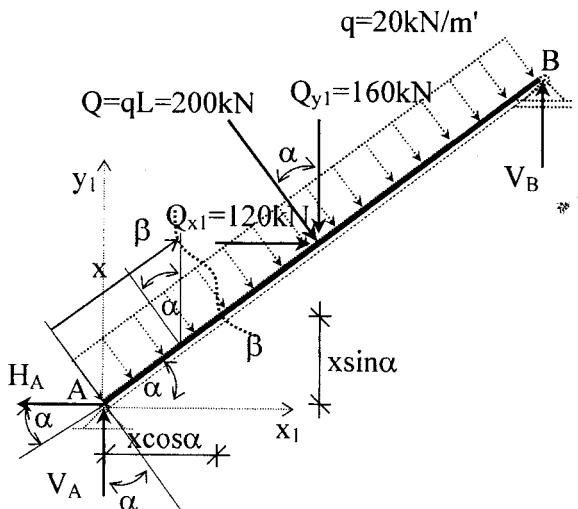
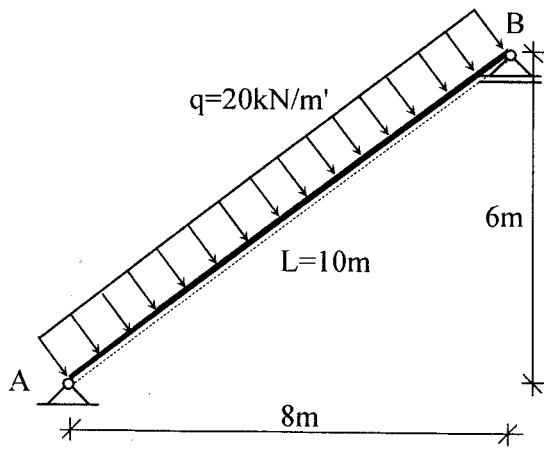
$$M_A = 0, \quad M_B = 0,$$

$$T_A = -T_B = V_A \cos \alpha = \frac{qL}{2} \cos \alpha = 100 \cdot \frac{8}{10} = 80\text{kN}.$$

$$-N_A = N_B = V_A \cos \alpha = 100 \cdot \frac{6}{10} = 60\text{kN}.$$



5.



$$\sin \alpha = \frac{6}{10} = 0.60$$

$$\cos \alpha = \frac{8}{10} = 0.80$$

Reakcije veza:

$$\sum X_1 = 0 \Rightarrow -H_A + Q \sin \alpha = 0, \quad (1)$$

$$\sum Y_1 = 0 \Rightarrow V_A + V_B - Q \cos \alpha = 0, \quad (2)$$

$$\sum M_A = 0 \Rightarrow V_B \cdot 8 - Q \cdot 5 = 0, \quad (3) \quad \text{ili} \quad V_B \cos \alpha \cdot L - qL \cdot \frac{L}{2} = 0$$

$$(1) \Rightarrow H_A = 120 \text{ kN}, \quad \text{ili} \quad H_A = qL \sin \alpha,$$

$$(3) \Rightarrow V_B = \frac{5}{8} Q = 125 \text{ kN}, \quad \text{ili} \quad V_B = \frac{qL}{2 \cos \alpha},$$

$$(2) \Rightarrow V_A = Q \cos \alpha - V_B = 35 \text{ kN}, \quad \text{ili} \quad V_A = qL \cos \alpha - \frac{qL}{2 \cos \alpha}.$$

Analitički izrazi za moment savijanja, transverzalnu i aksijalnu silu u polju proste grede glase:

$$M_\beta = V_A x \cos \alpha + H_A x \sin \alpha - \frac{qx^2}{2},$$

$$T_\beta = V_A \cos \alpha + H_A \sin \alpha - qx,$$

$$N_\beta = H_A \cos \alpha - V_A \sin \alpha.$$

Položaj i vrednost maksimalnog momenta u polju AB:

$$\frac{dM}{dx} = T = V_A \cos \alpha - H_A \sin \alpha - qx_o = 0 \Rightarrow x_o = \frac{L}{2} = 5 \text{ m},$$

$$M_{\max} = 35 \frac{8}{10} \cdot 5 + 120 \frac{3}{5} \cdot 5 - \frac{20 \cdot 5^2}{2} = 250 \text{ kNm}.$$

Vrednosti momenata savijanja, transverzalnih sila i aksijalnih sila na početku i na kraju polja su:

$$M_A = 0, \quad M_B = 0.$$

$$T_A = 35 \cdot \frac{8}{10} + 120 \cdot \frac{3}{5} = 100 \text{kN},$$

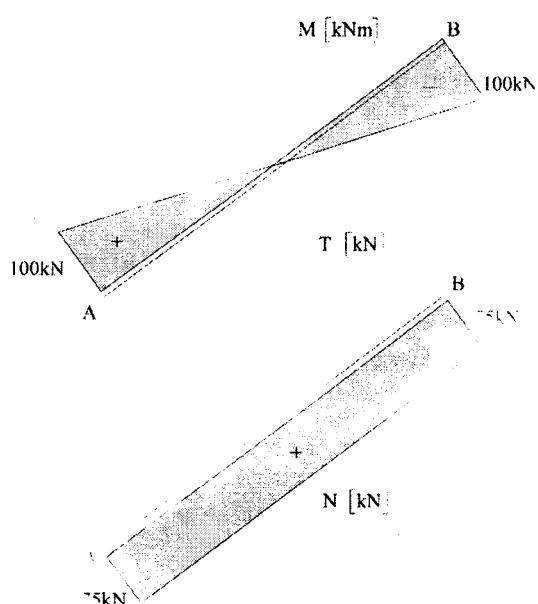
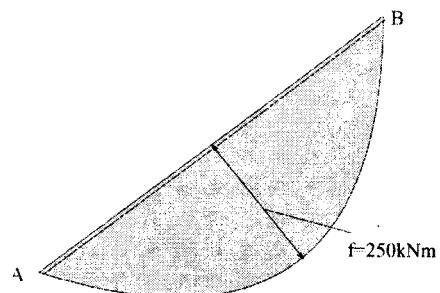
$$T_B = 35 \cdot \frac{8}{10} + 120 \cdot \frac{3}{5} - 20 \cdot 10 = -100 \text{kN},$$

$$N_A = 120 \cdot \frac{8}{10} - 35 \cdot \frac{6}{10} = 75 \text{kN},$$

$$N_B = 120 \cdot \frac{8}{10} - 35 \cdot \frac{6}{10} = 75 \text{kN}.$$

Maksimalni moment savijanja u polju izražen preko vrednosti kontinualnog opterećenja  $q$  i dužine grede  $L$  je:

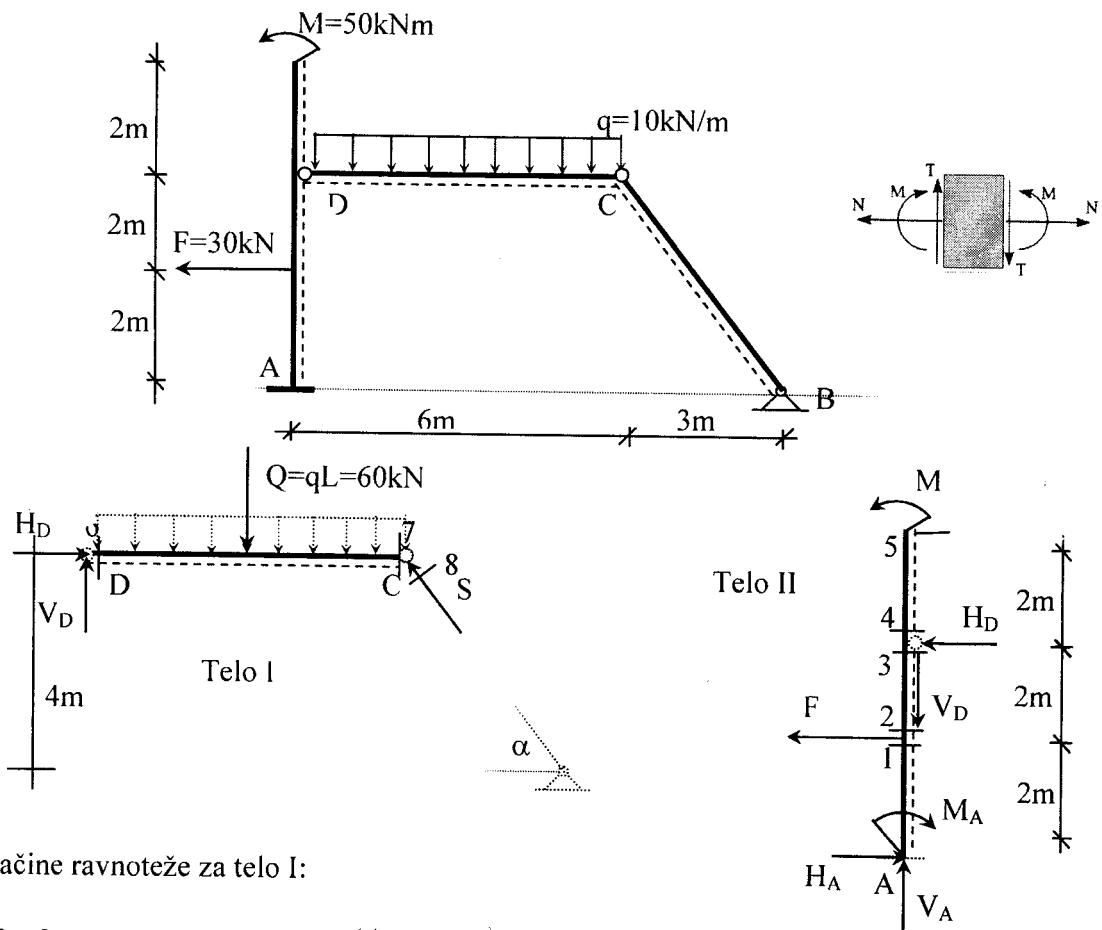
$$\begin{aligned} M_{\max} &= V_B \cos \alpha (L - x_o) - q(L - x_o) \frac{L - x_o}{2} \\ &= \frac{qL}{2 \cos \alpha} \cos \alpha \left( L - \frac{L}{2} \right) - \frac{q}{2} \left( L - \frac{L}{2} \right)^2 = \frac{qL^2}{8}. \end{aligned}$$



## Vežba br. 12b

### Sile u presecima. Razni zadaci

6. Odrediti komponente reakcija veza za dati sistem i opterećenje. Sračunati sile u presecima i nacrtati dijagrame M, T i N.



Jednačine ravnoteže za telo I:

$$\sum X = 0 \Rightarrow H_D - S \cos \alpha = 0, \quad (1)$$

$$\sum Y = 0 \Rightarrow V_D - Q + S \sin \alpha = 0, \quad (2)$$

$$\sum M_D = 0 \Rightarrow S \sin \alpha \cdot 6 - Q \cdot 3 = 0, \quad (3)$$

$$(1) \Rightarrow H_D = 22.5\text{kN}, \quad (3) \Rightarrow S = 37.5\text{kN}, \quad (2) \Rightarrow V_D = 30\text{kN}.$$

Jednačine ravnoteže za telo II:

$$\sum X = 0 \Rightarrow -H_D + H_A - F = 0, \quad (4)$$

$$\sum Y = 0 \Rightarrow -V_D + V_A = 0, \quad (5)$$

$$\sum M_A = 0 \Rightarrow -M_A + M + F \cdot 2 + H_D \cdot 4 = 0, \quad (6)$$

$$(4) \Rightarrow H_A = 52.5\text{kN}, \quad (6) \Rightarrow M_A = 200\text{kN}, \quad (5) \Rightarrow V_A = 30\text{kN}.$$

Sile u presecima

Momenti savijanja:

$$M_B = M_C = M_D = 0,$$

$$M_5 = M = 50 \text{ kNm} = M_4 = M_3,$$

$$M_1 = M + H_D \cdot 2 = 95 \text{ kNm} = M_2,$$

$$M_A = 200 \text{ kNm}.$$

$$f = \frac{qL^2}{8} = \frac{10 \cdot 6^2}{8} = 45 \text{ kNm}.$$

Transverzalne sile:

$$T_B = 0 = T_8,$$

$$T_7 = -S \sin \alpha = -30 \text{ kN},$$

$$T_6 = -S \sin \alpha + Q = 30 \text{ kN}, \text{ ili } T_6 = V_D = 30 \text{ kN},$$

$$T_5 = 0 = T_4,$$

$$T_3 = -H_D = -22.5 \text{ kN},$$

$$T_2 = -H_D = -22.5 \text{ kN},$$

$$T_1 = -H_D - F = -22.5 - 30 = -52.5 \text{ kN},$$

$$T_A = -H_A = -52.5 \text{ kN}.$$

Normalne sile:

$$N_8 = -S = -37.5 \text{ kN},$$

$$N_7 = -S \cos \alpha = -22.5 \text{ kN} = N_6,$$

$$N_5 = 0 = N_4.$$

$$N_3 = -V_D = -30 \text{ kN} = N_2 = N_1 = N_A.$$

